## GATE 2016 - A Brief Analysis

(Based on student test experiences in the stream of ME on 30th
January, 2016 - (Forenoon Session)
Section wise analysis of the paper

| Section Classification | $\mathbf{1}$ Mark | 2 Marks | Total No of <br> Questions |
| :--- | :---: | :---: | :---: |
| Engineering Mathematics | 5 | 5 | 10 |
| Engineering Mechanics | 2 | 2 | 4 |
| Strength of Materials | 4 | 4 | 8 |
| Design of Machine Elements | 0 | 1 | 1 |
| Theory of Machines | 1 | 2 | 3 |
| Vibrations | 1 | 1 | 2 |
| Fluid Mechanics | 3 | 4 | 7 |
| Thermal Science | 2 | 3 | 5 |
| Heat Transfer | 2 | 2 | 4 |
| Manufacturing Science | 4 | 4 | 8 |
| Industrial Engineering | 1 | 2 | 3 |
| Verbal Ability | 3 | 0 | 3 |
| Numerical Ability | 2 | 5 | 7 |
|  | 30 | 35 | 65 |

Type of Questions asked from each section

| Engineering Mechanics | Questions came from friction, equilibrium equations, <br> trusses. |
| :--- | :--- |
| Strength of materials | Questions came from simple stress and strain, stress - <br> strain graph, deflection of beam, Hardness test, Torsion of <br> circular shaft, Bending stress, Helical spring, Moment of <br> Inertia. |
| Machine design | Questions came from failure theory |
| Theory of machines | Questions came from Gear train, Gyroscope, fly wheel, <br> rotating masses. |
| Vibrations | Questions came from Resonance, single degree of <br> freedom of free vibration. |
| Fluid mechanics | Questions came from boundary layer, flow through pipe, <br> turbulent flow, stability of floating bodies, turbine, <br> manometry, |
| Thermal Science | Questions came from work done in non flow process, <br> thermodynamic relation, and thermodynamic system, <br> Vapour compression cycle, regeneration vapour cycle. |
| Heat transfer | Questions came from radiation, unsteady heat conduction. |
| Manufacturing Science | Questions came from riser design, rolling, casting, <br> welding, non traditional machining process, cutting tool. |
| Industrial Engineering | Questions came from LPP, Inventory |

## Questions from the Paper <br> General Aptitude

1. Despite the new medicine's $\qquad$ in treating diabetes, it is not $\qquad$ widely.
(A) effectiveness - prescribed
(B) availability - used
(C) prescription - available
(D) acceptance - proscribed

## Key: (A)

2. If $\mathrm{q}^{-\mathrm{a}}=\frac{1}{\mathrm{r}}, \mathrm{r}^{-\mathrm{b}}=\frac{1}{\mathrm{~s}}$ and $\mathrm{s}^{-\mathrm{c}}=\frac{1}{\mathrm{q}}$, then the value of a.b.c is $\qquad$ .
(A) $(\mathrm{rqs})^{-1}$
(B) 0
(C) 1
(D) $r+q+s$

Key: (C)
Exp: $\quad q^{-a}=\frac{1}{r}, r^{-b}=\frac{1}{s}, s^{-c}=\frac{1}{q}$
$q^{a}=r, r^{b}=s, s^{c}=q$
$\mathrm{r}=\mathrm{q}^{\mathrm{a}}=\left(\mathrm{s}^{\mathrm{c}}\right)^{\mathrm{a}}=\mathrm{s}^{\mathrm{ac}}$
$\mathrm{s}=\mathrm{r}^{\mathrm{b}}=\left(\mathrm{s}^{\mathrm{ac}}\right)^{\mathrm{b}}=\mathrm{s}^{\mathrm{abc}} \Rightarrow \mathrm{abc}=1$
3. Leela is older than her cousin Pavithra, Pavithra's brother shiva is older than Leela when Pavithra \& Shiva are visiting Leela, all the three like to play chess together, Pavithra wins more often than Leela does.
Which is true on the basis of above statement?
(A) When Shiva plays chess with Leela and Pavithra, he often loses
(B) Leela is oldest of the three
(C) Shiva is better chess player than Pavithra
(D) Pavithra is the youngest of the three.

Key: (D)
4. Michel lives 10 km away from where I live, Ahmed lives 5 km and Susan lives 7 km away from where I live. Arun is farther away then Ahmed, but closer than Susan from where I live. From the information provided, what is one possible distance (in km ) at which of I live from Arun
(A) 3.1
(B) 4.99
(C) 6.02
(D) 7.01

Key: (C)
5. Mount Everest is
(A) the highest peak in the world.
(B) highest peak in the world
(C) one of highest peak in the world
(D) one of the highest peak in the world.

## Key: (A)

6. Policemen asked the victim of a theft, "What did you $\qquad$ ?"
(A) loose
(B) lose
(C) loss
(D) louse

Key: (B)

[^0]
## Technical

1. An ideal gas undergoes a reversible process in which the pressure varies linearly with volume. The condition at the start (subscript 1) and at the end (subscript 2) of the process with usual notations are $\mathrm{P}_{1}=100 \mathrm{kPa}, \mathrm{V}_{1}=0.2 \mathrm{~m}^{3}$ and $\mathrm{P}_{2}=200 \mathrm{kPa}, \mathrm{V}_{2}=0.1 \mathrm{~m}^{3}$ and the gas constant $\mathrm{R}=$ $0.275 \mathrm{~kJ} / \mathrm{kg}-\mathrm{K}$. The magnitude of work required for the process (in kJ ) is $\qquad$ -.
Key: 22.5
Exp: Pressure varies linearly with volume.
$\mathrm{P}=\mathrm{a}+\mathrm{bv}$.
$P_{1}=a+b v$
$\Rightarrow 100=a+b \times 0.2$
$P_{2}=a+b v_{2}$
$\Rightarrow 200=a+b \times 0.1$
Solving (1) \& (2)
$-100=0.1 b \Rightarrow b=\frac{100}{0.1}=-1000$
$b=-1000$
Substituting in any of the equations to get ' $a$ '.

$$
\begin{aligned}
& \Rightarrow 100=\mathrm{a}+(-1000 \times 0.2) \Rightarrow \quad \mathrm{a}=300 \\
& \begin{aligned}
\therefore \mathrm{W} & =\int_{1}^{2} \mathrm{pdv}=\int_{1}^{2}(\mathrm{a}+\mathrm{bv}) \mathrm{dv} \\
& =\int_{1}^{2}(300-1000 \mathrm{v}) \mathrm{dv}=300\left(\mathrm{v}_{2}-\mathrm{v}_{1}\right)-1000\left(\frac{\mathrm{v}_{2}^{2}-\mathrm{v}_{1}^{2}}{2}\right) \\
& =300(0.1-0.2)-500\left(\frac{(0.1)^{2}-(0.2)^{2}}{2}\right)=-30-(-7.5)=-22.5 \mathrm{~kJ}
\end{aligned}
\end{aligned}
$$

$\therefore$ Magnitude of work required is 22.5 kJ .
2. For water $25^{\circ} \mathrm{C}, \mathrm{dP}_{\mathrm{s}} / \mathrm{dT}_{\mathrm{s}}=0.189 \mathrm{kPa} / \mathrm{K}\left(\mathrm{p}_{\mathrm{s}}\right.$ is the saturation pressure in kPa and T is the saturation temperature in K ) and the specific volume of dry saturated vapour is $43.38 \mathrm{~m}^{3} / \mathrm{kg}$. Assume that the specific volume of liquid is negligible in comparison with that of vapour. Using the Clausius-Clapeyron equation, estimate the enthalpy of evaporation of water at $25^{\circ} \mathrm{Cin}(\mathrm{kJ} / \mathrm{kg})$ is $\qquad$ —.

Key: 2443.24
Exp: $\quad \frac{\mathrm{dP}_{\mathrm{s}}}{\mathrm{dT}_{\mathrm{s}}}=\frac{\mathrm{h}_{\mathrm{fg}}}{\mathrm{T}_{\mathrm{s}}\left(\mathrm{v}_{\mathrm{g}}-v_{\mathrm{f}}\right)} \Rightarrow 0.189=\frac{\mathrm{h}_{\mathrm{fg}}}{(25+273)(43.38-0)}$

$$
\Rightarrow \mathrm{h}_{\mathrm{fg}}=2443.248 \mathrm{~kJ} / \mathrm{kg}
$$

3. Oil (Kinematic Viscosity, $v_{\text {oil }}=1 \times 10^{-5} \mathrm{~m}^{2} / \mathrm{s}$ ) flows through a pipe of 0.5 m diameter with a velocity of $10 \mathrm{~m} / \mathrm{s}$ (kinematic viscosity, $v_{\mathrm{w}}=0.89 \times 10^{-6} \mathrm{~m}^{2} / \mathrm{s}$ ) is flowing through a model pipe of diameter 20 mm . For satisfying the dynamic similarity, the velocity of water (in $\mathrm{m} / \mathrm{s}$ ) is $\qquad$ _.

Key: 22.25
Exp: Dynamic similarity
$(\mathrm{Re})_{\text {oil }}=(\mathrm{Re})_{\text {water }}$
$\left(\frac{\mathrm{ud}}{v}\right)_{\text {oil }}=\left(\frac{\mathrm{ud}}{v}\right)_{\text {water }}$
$\frac{0.5 \times 10}{1 \times 10^{-15}}=\frac{0.02 \times \mathrm{u}_{\text {water }}}{0.89 \times 10^{-6}}$
$\mathrm{u}_{\text {water }}=22.25 \mathrm{~m} / \mathrm{s}$
4. A 300 mm thick slab is being cold rolled using roll of diameter 600 mm . If co-efficient of friction is 0.08 , the maximum possible reduction (in mm ) is $\qquad$
Key: 1.92
Exp: $\quad(\Delta h)_{\text {max }}=\mu^{2} R$

$$
\begin{aligned}
& =(0.08)^{2} \times 300 \\
& =1.92 \mathrm{~mm}
\end{aligned}
$$

5. A hypothetical engineering stress-strain curve shown in the figure has 3 straight lines $\mathrm{PQ}, \mathrm{QR}, \mathrm{RS}$ with co-ordinate $\mathrm{P}(0,0), \mathrm{Q}(0.2,100), \mathrm{R}(0.6,140)$ and $\mathrm{S}(0.8,130)$. Q is the yield point, R is UTS point and S is fracture point.


The toughness of the material. (in $\mathrm{MJ} / \mathrm{m}^{3}$ ) is $\qquad$ .
Key: 0.85
Disclaimer - This paper analysis and questions have been collated based on the memory of some students who appeared in the paper and should be considered only as guidelines. GATEFORUM does not take any responsibility for the correctness of the same.

Exp: Toughness of material
Total area - [Area of $1+$ Area of $2+$ Area of $3+$ Area of 4]
$=\left(140 \times \frac{0.8}{100}\right)-\left[\left\{\frac{1}{2} \times 100 \times \frac{0.2}{100}\right\}+\left\{40 \times \frac{0.2}{100}\right\}+\left\{\frac{1}{2} \times 40 \times \frac{0.4}{100}\right\}+\left\{\frac{1}{2} \times 10 \times \frac{0.2}{100}\right\}\right]$
$=1.12-[0.1+0.08+0.08-0.01]$
$=0.85 \mathrm{MJ} / \mathrm{m}^{3}$

6. A steel ball of 10 mm diameter at 1000 K is required to be cooled to 350 K by immersing it in a water environment at 300 K . The convective heat transfer coefficient is $1000 \mathrm{~W} / \mathrm{m}^{2}-\mathrm{K}$. Thermal conductivity of steel is $40 \mathrm{~W} / \mathrm{m}-\mathrm{K}$. The time constant for the cooling process $\tau$ is 16 s . The time required (in-sec) to reach the final temperature is $\qquad$ .

Key: 42.22
Exp: Given
$\mathrm{d}=10 \mathrm{~mm}=0.01 \mathrm{~m}$
$\mathrm{t}_{\mathrm{i}}=1000 \mathrm{~K} ; \mathrm{t}=350 \mathrm{~K} ; \mathrm{t}_{\infty}=300 \mathrm{~K} ; \quad \mathrm{k}=40 \frac{\mathrm{~W}}{\mathrm{mK}} ; \mathrm{h}=1000 \frac{\mathrm{~W}}{\mathrm{~m}^{2} \mathrm{~K}} ; \tau_{\mathrm{th}}=16 \mathrm{~s}$
$\frac{t-t_{\infty}}{t_{i}-t_{\infty}}=e^{-\tau / \tau_{\text {th }}}$
$\ln \left(\frac{\mathrm{t}-\mathrm{t}_{\infty}}{\mathrm{t}_{\mathrm{i}}-\mathrm{t}_{\infty}}\right)=-\tau / \tau_{\mathrm{th}}$
$\ln \left(\frac{350-300}{1000-300}\right)=-\tau / 16$
$\tau=42.22 \mathrm{~s}$
7. A cylindrical job with diameter of 200 mm and height of 100 mm is to be cast using modulus method of riser design. Assume that the bottom surface of cylindrical riser does not contribute as cooling surface. If the diameter of the riser is equal to its height, then the height of the riser (in mm ) is
(A) 150
(B) 200
(C) 100
(D) 125 .

Key: (A)
Exp:
$\mathrm{d}_{\mathrm{c}}=200 \mathrm{~mm}$
$\mathrm{d}_{\mathrm{r}}=\mathrm{h}_{\mathrm{r}}$
C $=$ Casting
$\mathrm{h}_{\mathrm{c}}=100 \mathrm{~mm}$
$\mathrm{h}_{\mathrm{r}}=$ ?
$\mathrm{R}=$ Rises
$\mathrm{M}_{\mathrm{r}}=1.2 \mathrm{M}_{\mathrm{C}}$
$\left(\frac{\mathrm{V}}{\mathrm{S}}\right)_{\mathrm{r}}=1.2\left(\frac{\mathrm{~V}}{\mathrm{~S}}\right)_{\mathrm{C}}$
$\frac{\frac{\pi}{4} \mathrm{~d}_{\mathrm{r}}^{2} \mathrm{hr}}{\pi \mathrm{d}_{\mathrm{r}} \mathrm{h}_{\mathrm{r}}+\frac{\pi}{4} \mathrm{~d}_{\mathrm{r}}^{2}}=1.2 \frac{\frac{\pi}{4} \mathrm{~d}_{\mathrm{c}}^{2} \mathrm{~h}_{\mathrm{c}}}{\mathrm{d}_{\mathrm{c}} \mathrm{h}_{\mathrm{c}}+\frac{\pi}{4} \mathrm{~d}_{\mathrm{c}}^{2} \mathrm{x}^{2}}$
$=\frac{\mathrm{dr}^{2} h_{\mathrm{r}}}{4 \mathrm{~d}_{\mathrm{r}} \mathrm{h}_{\mathrm{r}}+\mathrm{d}_{\mathrm{r}}^{2}}=\frac{1.2 \mathrm{~d}_{\mathrm{c}}^{2} \mathrm{~h}_{\mathrm{c}}}{4 \mathrm{~d}_{\mathrm{c}} \mathrm{h}_{\mathrm{c}}+2 \mathrm{~d}_{\mathrm{c}}^{2}}$
$=\frac{\mathrm{h}_{\mathrm{r}}^{3}}{4 \mathrm{~h}_{\mathrm{r}}^{2}+\mathrm{h}_{\mathrm{r}}^{2}}=\frac{1.2 \times(200)^{2} \times 100}{4 \times 200 \times 100+2 \times 200^{2}}$
$=\frac{\mathrm{h}_{\mathrm{r}}}{5}=1.2 \times \frac{200 \times 10}{4 \times 100+2 \times 200}$
$=\mathrm{h}_{\mathrm{r}}=1.2 \times 5 \times \frac{200 \times 100}{4 \times 200}=125 \times 1.2=150$
8. A block of mass $m$ rests on an inclined plane and is attached by a string to the wall as shown in the figure. The coefficient of static friction between plane and the block is 0.25 . The string can withstand a maximum force of 20 N . The maximum value of mass (m) for which the string will not break and the block will be in static equilibrium is $\qquad$ kg.


Take $\cos \theta=0.8$ and $\sin \theta=0.6$
Acceleration due to gravity $\mathrm{g}=10 \mathrm{~m} / \mathrm{s}^{2}$
Key: 5

Disclaimer - This paper analysis and questions have been collated based on the memory of some students who appeared in the paper and should be considered only as guidelines. GATEFORUM does not take any responsibility for the correctness of the same.

Exp: $\quad F=\mu R=\mu m g \cos \theta$

$$
\begin{aligned}
& =0.25 \times \mathrm{m} \times 10 \times 0.8 \\
& =2 \mathrm{~m}
\end{aligned}
$$

For equilibrium,
$\mathrm{T}+\mathrm{F}=\mathrm{mg} \sin \theta$
$\Rightarrow 20+2 \mathrm{~m}=\mathrm{m} \times 10 \times 0.6 \Rightarrow \mathrm{~m}=5 \mathrm{~kg}$

9. A fluid (Prandtl No. $\operatorname{Pr}=1$ ) at 500 K flow over a flat plate of 1.5 m length maintained at 300 K and the velocity of the fluid is $10 \mathrm{~m} / \mathrm{s}$. Assuming kinetic viscosity $(\mathrm{v})=30 \times 10^{-6} \mathrm{~m}^{2} / \mathrm{s}$, the thermal boundary layer thickness (in mm ) at 0.5 m from the leading edge is $\qquad$
Key: 6.124
Exp: $\quad \operatorname{Pr}=1$

$$
\begin{aligned}
\operatorname{Re}_{\mathrm{x}} & =\frac{\mathrm{ux}}{v}=\frac{10 \times 0.5}{30 \times 10^{-6}} \\
& =166666.67 \\
& =1.67 \times 10^{5}
\end{aligned}
$$

Hydrodynamic boundary layer thickness

$$
\begin{aligned}
\delta_{\mathrm{h}_{\mathrm{x}}} & =\frac{5 \mathrm{x}}{\sqrt{\operatorname{Re}_{\mathrm{x}}}}=\frac{5 \times 0.5}{\sqrt{1.67 \times 10^{5}}} \\
& =6.124 \times 10^{-3} \mathrm{~m}
\end{aligned}
$$

If $\operatorname{Pr}=1$

$$
\begin{aligned}
\delta_{\mathrm{h}_{\mathrm{x}}} & =\delta_{\mathrm{T}_{\mathrm{x}}}=6.124 \times 10^{-3} \mathrm{~m} \\
& =6.124 \mathrm{~mm}
\end{aligned}
$$

$\therefore$ Thermal boundary layer thickness $=6.124 \mathrm{~mm}$.
10. A horizontal bar with a constant cross section is subjected to a loading as shown in the figure. The Young's modulus for the section AB and BC are 3 E and E respectively.


For the deflection C to be zero, the ratio of $\mathrm{P} / \mathrm{F}$ is $\qquad$ .

Key: 3
Exp: Since deflection of C is zero
$\Rightarrow \delta \ell_{\mathrm{AB}}=\delta \ell_{\mathrm{BC}}$
$\frac{\mathrm{P} \ell}{3 \mathrm{E} \times \mathrm{A}}=\frac{\mathrm{F} \ell}{\mathrm{E} \times \mathrm{A}}$
$\mathrm{P} / \mathrm{F}=3$
11. An infinitely long furnace of $0.5 \mathrm{~m} \times 0.4 \mathrm{~m}$ cross-section is shown in the figure below. Consider all surfaces of the furnace to be black. The top and bottom walls are maintained at temperatures $\mathrm{T}_{1}=\mathrm{T}_{3}=927^{\circ} \mathrm{C}$ and the side walls are at temperature $\mathrm{T}_{2}=\mathrm{T}_{4}=527^{\circ} \mathrm{C}$. The view factor $\mathrm{F}_{1-2}$ is 0.26 . The net radiation heat loss or gain on side 1 is $\qquad$ W/m


Key: 12265.34
Exp: $\quad \sigma=5.67 \times 10^{-8} \mathrm{~W} / \mathrm{m}^{2} \mathrm{~K}^{4}$

$$
\begin{aligned}
\mathrm{T}_{1}= & 927+273=1200 \mathrm{~K} \\
\mathrm{~T}_{2}= & 527+273=800 \mathrm{~K} \\
\mathrm{Q}_{1-2} & =\sigma \mathrm{A}_{1} \mathrm{~F}_{1-2}\left(\mathrm{~T}_{1}^{4}-\mathrm{T}_{2}^{4}\right) \\
& =5.67 \times 10^{-8} \times(0.5 \times 1) \times 0.26\left(1200^{4}-800^{4}\right) \\
& =12265.344 \mathrm{~W} / \mathrm{m}
\end{aligned}
$$


12. A simply supported beam of length 3 L is subjected to the loading shown in the Figure. It is given that $\mathrm{P}=1 \mathrm{~N}, \mathrm{~L}=1 \mathrm{~m}$ and Young's modulus $=200 \mathrm{GPa}$, The cross-sectional is a square with dimension $10 \mathrm{~mm} \times 10 \mathrm{~mm}$. The bending stress in $(\mathrm{Pa})$ at the point A located at the top surface of the beam at a distance of 1.5 L , from the left end is $\qquad$ .


Key: 0
$\overline{\text { Disclaimer - This paper analysis and questions have been collated based on the memory of some students who appeared in the }}$ paper and should be considered only as guidelines. GATEFORUM does not take any responsibility for the correctness of the same.

Exp:


Taking moment about B
$\Sigma \mathrm{M}_{\mathrm{B}}=0 \Rightarrow \mathrm{R}_{\mathrm{A}} \times 3 \mathrm{~L}+\mathrm{P} \times 2 \mathrm{~L}-\mathrm{PL}=0 \Rightarrow \mathrm{R}_{\mathrm{A}}=-\mathrm{P} / 3$
$\Sigma \mathrm{F}_{\mathrm{y}}=0 \Rightarrow \mathrm{R}_{\mathrm{B}}+\mathrm{R}_{\mathrm{A}}=0 \Rightarrow \mathrm{R}_{\mathrm{B}}=\mathrm{P} / 3$
Taking moment about A
$\Sigma \mathrm{M}_{\mathrm{A}}=0$
$\mathrm{R}_{\mathrm{A}} \times 1.5 \mathrm{~L}+0.5 \mathrm{PL}=\mathrm{M}_{\mathrm{A}} \quad$ (Assuming $\mathrm{M}_{\mathrm{A}}$ anticlockwise)
$\Rightarrow-\frac{\mathrm{P}}{3} \times 1.5+0.5 \mathrm{PL}=\mathrm{M}_{\mathrm{A}}$
$\Rightarrow \mathrm{M}_{\mathrm{A}}=0$
we know, $\frac{M}{I}=\frac{\sigma_{\mathrm{b}}}{\mathrm{y}} \Rightarrow \sigma_{\mathrm{b}} \quad($ Bending stress $)=0 \quad$ since, $\mathrm{M}_{\mathrm{A}}=0$
13. A two member truss PQR is supporting a load W . The axial forces in member PQ and QR are respectively.
(A) 2 W tensile and $\sqrt{3} \mathrm{~W}$ compressive
(B) $\sqrt{3} \mathrm{~W}$ tensile and 2 W compressive
(C) $\sqrt{3} \mathrm{~W}$ compressive and 2 W tensile
(D) 2 W compressive and $\sqrt{3} \mathrm{~W}$ tensile

Key: (B)
Exp: F.B.D of point Q

$\Sigma \mathrm{F}_{\mathrm{x}}=0$
$\Rightarrow \mathrm{F}_{\mathrm{PQ}}+\mathrm{F}_{\mathrm{QR}} \sin 60=0$
$\Sigma \mathrm{F}_{\mathrm{y}}=0$
$\Rightarrow \mathrm{F}_{\mathrm{QR}} \cos 60+\mathrm{W}=0$
$\Rightarrow \mathrm{F}_{\mathrm{QR}}=\frac{-\mathrm{W}}{\cos 60} \Rightarrow \mathrm{~F}_{\mathrm{QR}}=2 \mathrm{~W}$ (compressive)
From equation (1)

$\mathrm{F}_{\mathrm{PQ}}-2 \mathrm{~W} \sin 60=0$
$\Rightarrow \mathrm{F}_{\mathrm{PQ}}=2 \mathrm{~W} \times \frac{\sqrt{3}}{2}=\sqrt{3} \mathrm{~W}$ (Tensile)
14. In an orthogonal cutting process the tool used has rake angle of zero degree. The measured cutting force and thrust force are $500 \mathrm{~N} \& 250 \mathrm{~N}$, respectively. The coefficient of friction between tool and chip is $\qquad$ .
Key: 0.5
Exp: $\quad \operatorname{Tan}(\beta-\alpha)=\frac{\mathrm{F}_{\mathrm{t}}}{\mathrm{F}_{\mathrm{C}}}$
$\operatorname{Tan}(\beta-0)=\frac{250}{500}$
$\operatorname{Tan} \beta=1 / 2=0.5$
15. For a floating body, buoyant force acts at the
(A) Centroid of floating body,
(B) C.G of body
(C) Centroid of fluid vertically below the body
(D) Centroid of displaced fluid

Key: (D)
16. Consider two Hydraulic turbines having identical specific speed and effective head at inlet. If the speed ratio $\left(N_{1} / N_{2}\right)$ of the two turbines is 2 , then the ratio of respective power $\left(P_{1} / P_{2}\right)$ is $\qquad$ .
Key: 8
Exp: $\quad \frac{\mathrm{P}_{1}}{\mathrm{P}_{2}}=\left(\frac{\mathrm{N}_{1}}{\mathrm{~N}_{2}}\right)^{3}=2^{3}=8$
17. The spring constant of a helical compression spring doesn't depend on
(A) Coil diameter
(B) Material strength
(C) Number of active turns
(D) Wire diameter

Key: (B)
Exp: For Helical compression spring.
Deflection, $\delta=\frac{64 \omega \mathrm{R}^{3} \mathrm{n}}{\mathrm{Gd}^{4}}$
Stiffness, or spring constant $=\frac{W}{\delta}=\frac{W}{\frac{64 \mathrm{WR}^{3} \mathrm{n}}{\mathrm{Gd}^{4}}}=\frac{\mathrm{Gd}^{4}}{64 \mathrm{R}^{3} \mathrm{n}}$
From the above formula we can say that spring constant depends on coil diameter (D), wire diameter (d), No. of active turns ( n ) and modulus of rigidity ( G ) and is independent of material strength.
18. A single degree of freedom spring mass system with viscous damping has a spring constant of 10 $\mathrm{kN} / \mathrm{m}$. The system is excited by a sinusoidal force of amplitude 100 N . If the damping factor (ratio) is 0.25 , the amplitude of steady state oscillation at resonance is $\qquad$ mm.

Key: 20
Exp: $\quad \mathrm{k}=10 \times 10^{3} \mathrm{~N} / \mathrm{m}$
$\mathrm{A}=100 \mathrm{~N}$
$\xi=0.25$
M.F $=\frac{1}{\left(\left(1-\left(\frac{\omega}{\omega_{n}}\right)^{2}\right)^{2}+\left(2 \xi \cdot \frac{\omega}{\omega_{n}}\right)^{2}\right)^{1 / 2}} \Rightarrow \frac{\mathrm{~A}}{\mathrm{~F}_{0} / \mathrm{k}}=\frac{1}{\left(\left(\left(1-\frac{\omega}{\omega_{\mathrm{n}}}\right)^{2}\right)^{2}+\left(2 \xi \frac{\omega}{\omega_{\mathrm{n}}}\right)^{2}\right)^{1 / 2}}$
$\Rightarrow \mathrm{A}=\frac{100}{10 \times 10^{3} \times(2 \times 0.25)}=\frac{100}{5000}=20 \mathrm{~mm}$
19. $\operatorname{Max} z=15 x_{1}+20 x_{2}$

Subject to
$12 \mathrm{x}_{1}+4 \mathrm{x}_{2} \geq 36$
$12 \mathrm{x}_{1}-6 \mathrm{x}_{2} \leq 24$
$\mathrm{x}_{1}, \mathrm{x}_{2} \geq 0$
The above LPP has
(A) Infeasible solution,
(B) Unbounded solution
(C) Alternative optimum solution
(D) Degenerate solution

Key: (B)
Exp: $\quad$ Max $z=15 x_{1}+20 x_{2}$
Subject to
$12 \mathrm{x}_{1}+4 \mathrm{x}_{2} \geq 36$
$12 \mathrm{x}_{1}-6 \mathrm{x}_{2} \leq 24$
$\mathrm{x}_{1}, \mathrm{x}_{2} \geq 0$
Since, there is no limitation of boundary for the feasible region therefore, the LPP has unbounded solution.

20. The figure shows Cross-section of a beam subjected to bending. The are moment of inertia (in $\mathrm{mm}^{4}$ ) of this cross-section about its base is $\qquad$ .

(All dimensions are in mm )

Disclaimer - This paper analysis and questions have been collated based on the memory of some students who appeared in the paper and should be considered only as guidelines. GATEFORUM does not take any responsibility for the correctness of the same.

Key: 1875.63
Exp: By using parallel axis theorem
Area moment of inertia about its base
$=\left\{\frac{10 \times 10^{3}}{12}+\left(10 \times 10 \times 5^{2}\right)\right\}-\left\{\left(\frac{\pi}{64} \times 8^{4}\right)+\pi \times 4^{2} \times 5^{2}\right\}$
$=3333.33-1457.69=1875.63 \mathrm{~mm}^{4}$
21. The annual demand for an item is 10,000 units. The unit cost is Rs. 100 and inventory carrying charges is $14.4 \%$ of unit cost per annum. The cost of one procurement is Rs 2000.
The time between two consecutive orders to meet the above demand is month(s).
Key: 2
Exp: $\quad \mathrm{D}=10,000, \mathrm{Cu}=$ Rs $100, \mathrm{C}_{\mathrm{h}}=0.144 \times \mathrm{Cu}, \mathrm{C}_{\mathrm{o}}=$ Rs 2,000 .
$\mathrm{Q}^{*}=\sqrt{\frac{2 \mathrm{DC}_{0}}{\mathrm{C}_{\mathrm{h}}}}=\sqrt{\frac{2 \times 10,000 \times 2000}{0.144 \times 100}}=1666.67$ units
We know $\mathrm{T}=\frac{\mathrm{Q}^{*}}{\mathrm{D}}=\frac{1666.67}{10,000}=0.1667$ years $=2$ months

22. The Non traditional machining process that essentially, requires vacuum is
(A) Electron beam machining
(B) Electro-chemical machining
(C) Electro chemical discharge machining
(D) Electrical discharge machining.

Key: (A)
23. A rigid ball of weight 100 N is suspended with the help of a string. The ball is pulled by a horizontal force F such that the string makes and angle $30^{\circ}$ with the vertical. The magnitude of force $\mathrm{F}(\mathrm{in} \mathrm{N}$ ) is $\qquad$ _.
Key: 57.735
Exp: Applying Lami's theorem
$\frac{\mathrm{T}}{\sin 90}=\frac{100}{\sin (90+30)}=\frac{\mathrm{F}}{\sin (180-30)}$ $\mathrm{F}=57.735 \mathrm{~N}$

24. Which of the following statement are true regarding Heat and work?
(i) Boundary phenomena
(ii) Exact differential
(iii) Path function
(A) Both (i) \& (ii),
(B) Both (i) \& (iii),
(C) Both (ii) \& (iii)
(D) Only (iii).

Key: (B)
25. Jominy test is used to find
(A) Young's Modulus
(B) Hardenability
(C) Yield stress
(D) Thermal conductivity

Key: (B)
26. The cross-section of two hollow bars made of same material and concentric circles are shown in the figure. It is given that $r_{3}>r_{1}$ and $r_{4}>r_{2}$ and that of area of cross-section are same. $\mathrm{J}_{1}$ and $\mathrm{J}_{2}$ are torsional rigidity of the bars left and right
 respectively. The ratio of $\mathrm{J}_{2} / \mathrm{J}_{1}$ is

(A) $>1$
(B) $<0.5$
(C) $=1$
(D) between $0.5 \& 1$.

Key: (A)
Exp: $\quad$ Torsionalrigidity $(\mathrm{J})=$ shear modulus $\times$ polar moment of inertia
$\frac{\mathrm{J}_{2}}{\mathrm{~J}_{1}}=\frac{\mathrm{G}_{1} \times \frac{\pi}{2}\left(\mathrm{r}_{4}^{4}-\mathrm{r}_{3}^{4}\right)}{\mathrm{G}_{2} \times \frac{\pi}{2}\left(\mathrm{r}_{2}^{4}-\mathrm{r}_{1}^{4}\right)}=\frac{\mathrm{r}_{4}^{4}-\mathrm{r}_{3}^{4}}{\mathrm{r}_{2}^{4}-\mathrm{r}_{1}^{4}} \quad\left(\because \mathrm{G}_{1}=\mathrm{G}_{2}\right)$
Area of C/S is same
$\Rightarrow \pi\left(\mathrm{r}_{2}^{2}-\mathrm{r}_{1}^{2}\right)=\pi\left(\mathrm{r}_{4}^{2}-\mathrm{r}_{3}^{2}\right)$
$\Rightarrow \mathrm{r}_{2}^{2}-\mathrm{r}_{1}^{2}=\mathrm{r}_{4}^{2}-\mathrm{r}_{3}^{2}$
From equation (1)
$\frac{\mathrm{J}_{2}}{\mathrm{~J}_{1}}=\frac{\left(\mathrm{r}_{4}^{2}+\mathrm{r}_{3}^{2}\right)\left(\mathrm{r}_{4}^{2}-\mathrm{r}_{3}^{2}\right)}{\left(\mathrm{r}_{2}^{2}+\mathrm{r}_{1}^{2}\right)\left(\mathrm{r}_{2}^{2}-\mathrm{r}_{1}^{2}\right)}=\frac{\left(\mathrm{r}_{4}^{2}+\mathrm{r}_{3}^{2}\right)}{\left(\mathrm{r}_{2}^{2}+\mathrm{r}_{1}^{2}\right)}$
but $\mathrm{r}_{4}^{2}+\mathrm{r}_{3}^{2}>\mathrm{r}_{2}^{2}+\mathrm{r}_{1}^{2} \quad$ since, $\mathrm{r}_{3}>\mathrm{r}_{1}$ and $\mathrm{r}_{4}>\mathrm{r}_{2}$
$\Rightarrow \frac{\mathrm{J}_{2}}{\mathrm{~J}_{1}}>1$
27. The part of a gating system which regulates the rate of pouring of molten metal is $\qquad$ -
(A) Pouring basin
(B) Runner
(C) Choke
(D) Ingate

Key: (C)
28. A Cantilever beam having square cross-section of side a is subjected to an end load. If a is increased by $19 \%$, the tip deflection decreased approximately by
(A) $19 \%$
(B) $29 \%$
(C) $41 \%$
(D) $50 \%$

Key: (D)

Exp: $\quad \delta=\frac{\mathrm{p} \ell^{3}}{3 \mathrm{EI}} \Rightarrow \delta \alpha \frac{1}{\mathrm{I}} \Rightarrow \delta \alpha \frac{1}{\mathrm{a}^{4}}$
$\frac{\delta_{1}}{\delta_{2}}=\frac{\mathrm{a}_{2}^{4}}{\mathrm{a}_{1}^{4}}$ where, $\mathrm{a}_{2}=1.19 \mathrm{a}$,
$\Rightarrow \frac{\delta_{1}}{\delta_{2}}=(1.19)^{4} \Rightarrow \delta_{2}=\frac{\delta_{1}}{(1.19)^{4}}=0.5 \delta_{1}$
So, deflection decrease by $50 \%$
29. The instantaneous stream wise velocity of a turbulent flow is given as follows $\mathrm{u}(\mathrm{x}, \mathrm{y}, \mathrm{z}, \mathrm{t})=\overline{\mathrm{U}}(\mathrm{x}, \mathrm{y}, \mathrm{z})+\mathrm{U}^{\prime}(\mathrm{x}, \mathrm{y}, \mathrm{z}, \mathrm{t})$.
The time average of fluctuating velocity $\mathrm{v}^{\prime}(\mathrm{x}, \mathrm{y}, \mathrm{z}, \mathrm{t})$ is
(A) $\frac{\mathrm{U}^{\prime}}{2}$
(B) $\frac{-\overline{\mathrm{U}}}{2}$
(C) 0
(D) $\frac{\overline{\mathrm{U}}}{2}$

Key: (C)
Exp: $\quad$ Instantaneous velocity $=u(x, y, z, t)=\overline{\mathrm{U}}(\mathrm{x}, \mathrm{y}, \mathrm{z})+\mathrm{u}^{\prime}(\mathrm{x}, \mathrm{y}, \mathrm{z}, \mathrm{t})$
Time average of fluctuating velocity should be zero.
30. Under optimal condition of the process the temperature experienced by a copper work piece in fusion welding, brazing and soldering are such that
(A) $T_{\text {welding }}>T_{\text {soldering }}>T_{\text {brazing }}$
(B) $T_{\text {Soldering }}>T_{\text {Welding }}>T_{\text {Brazing }}$
(C) $T_{\text {brazing }}>T_{\text {welding }}>T_{\text {soldering }}$
(D) $T_{\text {welding }}>T_{\text {brazing }}>T_{\text {soldering }}$

Key: C


[^0]:    Disclaimer - This paper analysis and questions have been collated based on the memory of some students who appeared in the paper and should be considered only as guidelines. GATEFORUM does not take any responsibility for the correctness of the same.

